## Paper / Subject Code: 41901 / Process Equipment Design (PED)

1T00517 - B.E.(CHEMICAL)(Sem VII) (CBSGS) / 41901 - Process Equipment Design (PED)

	(3 hours) May	kimum marks: 80
ľ	N.B.	
1	1. Question No. 1 is <b>compulsory</b> .	
2	2. Attempt any <b>three</b> questions out of remaining <b>five</b> questions.	
3	3. Assume any suitable data wherever required.	
4	4. Draw figures wherever needed.	
1	1. Write short notes on any <b>four</b> .	(20)
	a) Expansion bellows	
	b) Multilayered high pressure vessels	
	c) Types of downcomers	
	d) Rotary drum filter	
	e) Entrainment separators	
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	2. Design a Fixed-tubesheet heat exchanger for the following data:	Service A Barrell
	Shell side:	
	Design pressure = 0.5 N/mm <sup>2</sup>	
	Permissible stress for shell material, Carbon steel = 95 N/mm <sup>2</sup>	
	Tube side:	
	Number of tubes = 60	2222
	Tube outside diameter = 20 mm	1000 8.
	Design pressure of tube side fluid = 2.5 N/mm <sup>2</sup>	
	Permissible stress of tube material = 120 N/mm <sup>2</sup>	2000 B
	Tube pitch = Square	8,00,
	Channel flange	7. 25. Contraction of the contra
	Flange material of construction – carbon steel	×
	Permissible stress for flange material = 95 N/mm <sup>2</sup>	
	Permissible stress for bolt material = 140 N/mm <sup>2</sup>	
	Joint with tube sheet – Ring facing	
	Ring gasket width = 18 mm	
	Gasket factor = 5.5	
	Gasket seating stress = 126 N/ mm <sup>2</sup>	
	Permissible stress for flange and tube sheet material= 100 N/mm <sup>2</sup>	
	Design	(0)
	Shell ( diameter and thickness)	(8)
	Flange between channel and tubesheet	(6)
	Tube sheet thickness	(4)
	channel cover thickness	(2)
×3	3. (a) Write the design procedure of calculating shell wall thickness of a ta	ıll column. Design should
	include determination of various stresses, calculation of resultant longitudna	_
77 (	stress:	(12)
$\wedge$	(b) Write notes on column internals for a packed column.	(8)

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I) II)

4.Design a calendria type evaporator with the following data assuming that it has baffle for entrainment separation.

= external pressure 0.1 N/mm<sup>2</sup> Evaporator drum under vacuum = 3,000 kg/hr Amount of water to be evaporated Heating surface required  $= 250 \text{ m}^2$  $= 0.2 \text{ N/mm}^2$ Steam pressure Density of liquid  $= 1000 \text{ kg/m}^3$ Density of vapor  $= 0.085 \text{ kg/m}^3$ Effective Tube length = 1500 mm Tube outside diameter = 100 mm Tube thickness = 1.5 mm Tubes laid on triangular pitch Assume downtake pipe as 50% of the total tube cross sectional area Permissible stress for evaporator material = 98 N/ mm<sup>2</sup> Poisson's ratio Modulus of elasticity for carbon steel =  $2 \times 10^5 \text{N/mm}^2$ Modulus of elasticity for brass  $= 9.5 \times 10^4 \text{ N/mm}^2$ Design the (i) Calendria(Diameter and thickness) (8)(ii) Vapor drum (Diameter and thickness) (8)(iii)Top torispherical head (4)5.(a) Show the symbols for the following components (i) Needle valve (ii) Centrifugal pump (iii) Filter press (iv) Tray drier (v) Jacketed vessel (5) (b)Write notes on i) Block diagram ii) Piping and Instrumentation Diagram (10)

- (a) Estimate the optimum pipe diameter for a water flow rate of 12 kg/s at 20°C. Carbon steel pipe is used. Density of water 1000 kg / m³ Viscosity of water at 20°C is 1.1 x 10 -3 Ns/m². Also find whether flow is laminar or turbulent.
- 6. A high pressure compound cylinder consists of an inner tube of inner diameter 20 mm and outer diameter of 40 mm. On it is shrunk fit a tube of external diameter 60 mm. The shrink fit should be so done that the contact pressure at the two surfaces do not exceed 1.602 N/mm². The cylinder is then subjected to an internal pressure of 10 N/mm². Calculate the original dimensions of the tubes and plot the stress distribution.

Coefficient of thermal expansion =  $12 \times 10^{-6}$  per  $^{0}$ C, E =  $200 \times 10^{3}$  N/mm<sup>2</sup>